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## THEORETICAL FRAMEWORK FOR THE EFFICIENCY EVALUATION OF COAL MINING MACHINES

### Introduction

The operating practice of coal mining machines shows [1–4] that their efficiency, especially in cutting complex structure coal seams of high cuttability in many ways depends on whether the cutter head design meets operating conditions, as well as economical [5], technological and engineering constraints [6, 7]. In this respect, designers need tools to calculate the load spectrum of a cutter head in different operating conditions of mining different cuttability coal. The load spectrum governs the limit torque of transmission mechanism, allowable torsion torque with regard to durability of transmission gears etc., i.e., the determinants of the above-listed constraints. First of all, it is necessary to determine structure and properties of coal seams, estimable using an aggregate index of equivalent coal cuttability  $A_{eqv}$  [8, 9], and to find mechanisms of their effect on pick failure.

Theoretical evaluation of production of energy resources is demandable to ensure sustainable fuel supply to power generating plants [9]. By now, Russian thermal power stations produce 17.9% of heat and electric energy [10]. Thus, it is critical to create theoretical framework for the optimal design of picks and cutter heads of coal mining machines to ensure required capacity of production faces in specific geological conditions.

### Coal mining machine capacity

The main economic efficiency criterion of a mining machine is the theoretical (estimated) capacity ( $Q_t$ , t/min) given by:

$$Q_t = B_{ww} H_{mt} V_{ft} \gamma_{coal}, \text{ t/min}, \quad (1)$$

where  $B_{ww}$  is the web width, m;  $H_{mt}$  is the mineable thickness of coal seam, m;  $V_{ft}$  is the feed rate, m/min;  $\gamma_{coal}$  is the density of coal, t/m<sup>3</sup>.

In this manner, the theoretical capacity is directly proportional to the mining machine feed rate in specific operating conditions. The feed rate is directly proportional to the depth of cut which is a key index of the cutter head efficiency. Thus, theoretical capacity of a mining machine is governed by the efficiency (capacity) of the cutter head of the machine.

Efficiency of a shearer is most often evaluated as a relation between the total intake power  $P_\Sigma$  and the capacity  $Q_t$  in the specific conditions of coal structure and cuttability assessed in aggregate by the index  $A_{eqv}$  (N/mm):

*Efficiency of coal mining machines in terms of their productivity and reliability depends on the conformity of cutter head designs with operating conditions and with a system of economical, technological and engineering constraints conditioned by load spectrum which influences the limiting moment of transmissions, allowable torsion torque of transmission with regard to gearing durability, etc. The system of constraints consists of economical constraints—pick replacement conditions, maximal pick productivity, allowable coal grade; technological constraints—roof support capacity, coal haulage capacity, gas criterion, continuous power of motor; as well as engineering constraints—maximal rate of feed of shearer, pulling power of feed assembly, maximal (stable) moment, radial overhang of picks. The constraints connected with the machine feed rate, pulling power and the maximal torsion torque (stable power) depend on breaking characteristics of coal seam. The aggregate evaluation of coal breaking characteristics is proposed to perform using the equivalent cuttability index of coal. The newly proposed index, as against the previously used cuttability index, tells more on the dynamics of cutting complex-structure coal seams with hard inclusions. The relations to determine theoretical productivity of coal mining machines, maximal torsion torque of motors, load variation factor of cutter heads with regard to their high-frequency and low-frequency components are presented. The authors exemplify determination of possible productivity and efficient application domain for shearers in case of technological and engineering constraints.*

**Keywords:** theoretical capacity, loading, coal breaking characteristics, cutter head, variation factor, operating conditions of shearer.

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$$P_\Sigma = f Q_t \text{ (at } A_{eqv} = \text{const)}. \quad (2)$$

The dependence of the type of (2) has been in use for a long time [11, 12]. At the known operating conditions of cutter head and characteristics of the medium to be cut, this relation defines the required power as a function of the preset capacity of a mining machine. Vice versa, at the known operating conditions of a mining machine, its elements (e.g., torque of cutting drive unit) or units of a longwall system (advance of support, capacity of armored face conveyor, etc.), this relation determines potential capacity in the preset conditions:

$$Q_t = f(A_{eqv}). \quad (3)$$

The dependence of the type of (3) is an operation constraint of a mining machine as a function of the coal cuttability  $A_{eqv}$ . For shearer (the most complex case against the plowing machines), all possible operation constraints are divided into three groups: economical, technological and engineering constraints. The economical constraints are the most essential: minimal allowable (economically expedient) capacity, allowable grade, replacement conditions of tools. These constraints (1E–3E in Fig. 1) and the analogous constraints (dust generation, energy intake) define the minimal capacity below which the use of the given machine model is assumed as unprofitable (lower bound of possible operation conditions for mining machines). Usually, these constraints are given by:  $Q_t \geq Q_{t.all}$  (where  $Q_{t.all}$  is the allowable capacity) and indirectly depend on  $A_{eqv}$ .

The technological constraints include the rate of feed, capacities of coal haulage machines and roof support, gas criterion and permanent output of motor (since the cycle duration and the motor duty cycle are governed by the operating condition of the longwall system) [13]. The technological constraints usually define the wanted upper bound of productivity (1T–3T in **Fig. 1**), i.e.,  $Q_t \geq Q_{t,tech}$  (where  $Q_{t,tech}$  is the theoretical capacity with regard to the technological constraints).

Alongside with  $A_{eqv}$  independence of the most constraints, there permanent output relation  $P_{perm} = f(A_{eqv})$  nonlinearly bounds (4T in **Fig. 1**) the possible productivity field from the right.

The engineering (design) constraints are the most essential and governed by the shearer design. These constraints first of all relate the maximal rate of feed, pulling power of feed assembly, allowable overhang of picks, maximal torque in the conditions of actual power supply at proper output\* (1D–4D in **Fig. 1**). The methods to determine these engineering constraints are in detail reviewed in [9]. The pulling power constraint and the maximal torque constraint (stable output) depend on coal breaking characteristics and should be evaluated in aggregate using the equivalent cuttability index  $A_{eqv}$ , which, unlike the previous cutting resistance index  $A_{cut}$ , stronger describes cutting dynamics in complex structure coal seams containing large solid inclusions and hard dirt bands.

These constraints, in the aggregate, shape a system of productivity constraints shown in **Fig. 1** in the coordinates  $Q_t(V_{tr}) - A_{eqv}$ . The efficient productivity domain of shearer lies inside the field of possible operating conditions *abcd* (**Fig. 1**). The operating conditions which fit the points inside this field are allowable for a given machine with regard to all three groups of constraints. The line *bc* characterizes the application range of the mining machine with respect to the index  $A_{eqv}$ , and the point *c* shows the allowable application domain.

According to the analysis [14], the constraints that govern potential productivity are, as a rule, the engineering constraints of the pulling power (for shearers to mine coal seams with a thickness more than 3 m and a dip angle more than 15°), moment (thin coal seams) or heat (heavy duty longwalling) of motors. The research shows [15] that the capacity of mining machines with a high power-to-weight ratio is also limited by durability of the drive elements as an increase in the drive capacity (at the same size, materials and gearing) results in a considerable decrease in the safety factor of the transmission elements subjected to higher loading.

The quantitative estimate of the influence exerted by the coal seam structure on the coal strength characteristics is also required as the latter govern, alongside with the other factors, the dynamic load of the mining machine motor [7, 8, 11, 16–18].

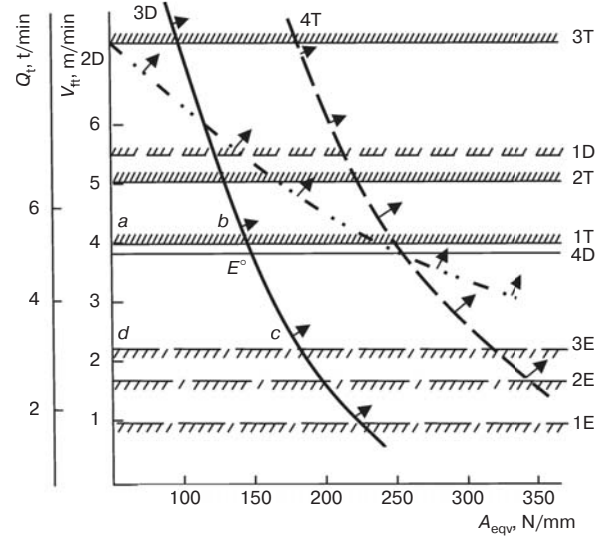
When a mining machine operates different capacity motors at the same time, the operation is constrained by the limiting moment tolerable by the strength of the transmission elements.

In the general case, this constraint is given by:

$$M_{max} = M_{av} + M_{dyn} \leq M_{all}, \quad (4)$$

where  $M_{max}$  and  $M_{all}$  are the current allowable maximal moments in transmission to cutter head, respectively;  $M_{av}$  is the average moment;  $M_{dyn}$  is the dynamic moment.

\*The technical literature widely uses the terms of 'stable moment' and 'stable output'.



**Fig. 1. Mining machine operation constraints: Economical – cutting tool replacement conditions (1E), minimum capacity (2E), allowable coal grade (E3); Technological – support capacity (1T), coal haulage capacity (2T), gas criterion (3T), permanent output of motor (4T); Engineering (Design) – maximal feed rate of shearer (1D), pulling force of feed assembly (2D), maximal (stable) moment (3D), radial overhang of pick (4D)**

The current maximal moment:

$$M_{max} = (F_{ch} + Z_{pick}(1 + 3v_{zpick}))D_{ch}/2, \text{ N}\cdot\text{m}, \quad (5)$$

where  $F_{ch}$  is the total cutting force of cutter head, N;  $Z_{pick}$  is the maximal average-pick cutting force of pick, N;  $v_{zpick}$  is the variation factor of the average-pick cutting force;  $D_{ch}$  is the diameter of the cutter head, m.

Using (4) and (5) allows the unknown constrain to be given by:

$$F_{ch.all} \leq (2M_{all}/D_{ch}) - Z_{pick}(1 + 3v_{zpick}), \quad (6)$$

where  $F_{ch.all}$  is the allowable circumferential force of the cutter head with respect to the strength of the transmission elements.

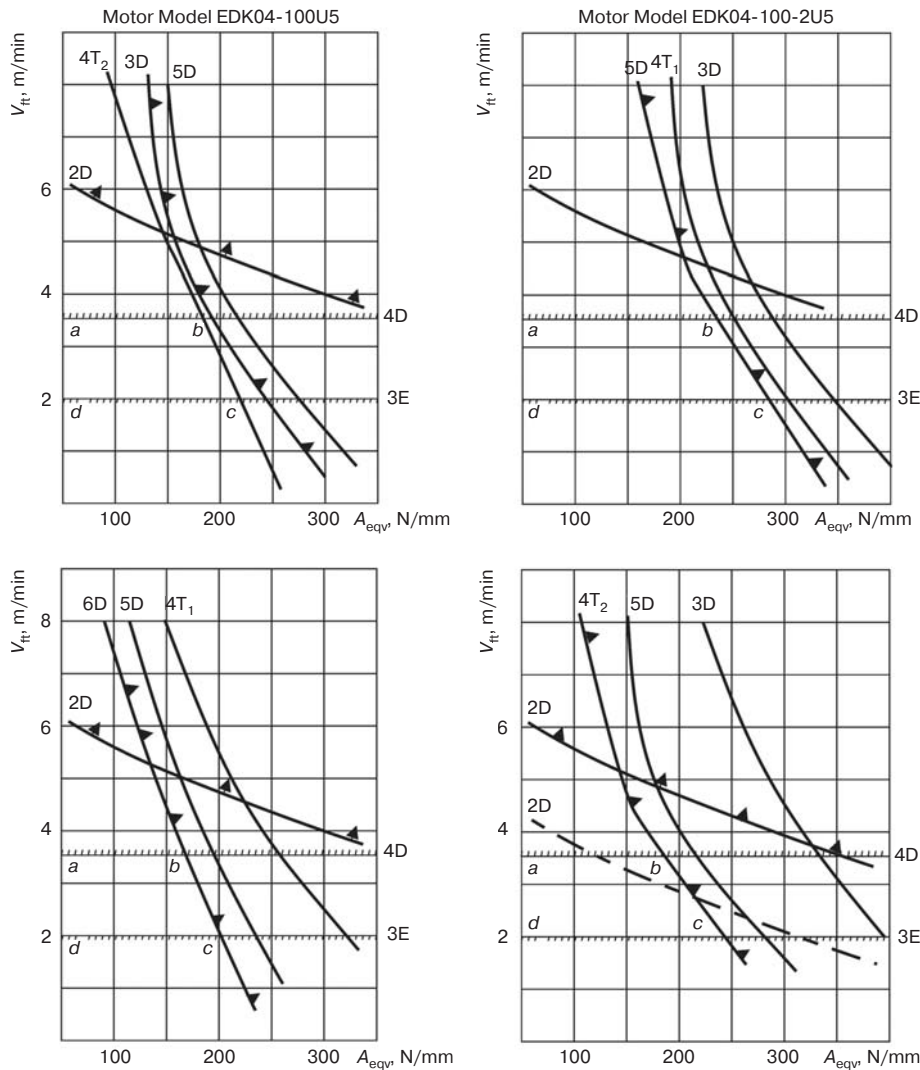
It is necessary to introduce the durability constraint of the most loaded element of transmission (usually, gear). The authors propose a relation to evaluate the allowable average torsion torque  $M_{tt.all}$  to agree with the equivalent moment in the strength design at the safety factor of 1 in case of the varying loading of the transmission:

$$M_{tt.all} = \frac{M_{eqv} K_{SF}^2 (1 - v_{tr}^{2.4})^3}{m_{eqv}} \sqrt[3]{\frac{T_d}{T_r}}, \quad (7)$$

where  $M_{eqv}$  and  $K_{SF}$  are, respectively, the estimated equivalent moment and safety factor of the highest loaded transmission gear;  $v_{tr}$  is the variation factor of loads;  $m_{eqv}$  is the coefficient of equivalence;  $T_d$  and  $T_r$  are the design and estimated durabilities, respectively:

$$v_{tr} = \sqrt{v_{1i}^2 + K_{gain}^2 (v_{2i}^2 + v_{3i}^2) + v_{4i}^2 + v_{5i}^2}, \quad (8)$$

where  $v_{1i}$ ,  $v_{4i}$ ,  $v_{5i}$  are the low-frequency components of the load variation factor;  $K_{gain}$  is the gain constant;  $v_{2i}$ ,  $v_{3i}$  are the high-frequency components of the load variation factor.



**Fig. 2. Potential productivity and efficient application domains of shearer at different constraints: Design constraints – pulling power (2D), maximal (stable) moment (3D), rate of feed (4D), durability of the highest loaded element of transmission (5D), durability of motor (6D); Technological constraints – continuous power at duty cycle of 25 and 40% (4T<sub>1</sub> and 4T<sub>2</sub>, respectively); Economical constraints – allowable grade of coal (3E)**

It follows from the formula (8) that the load spectrum on the cutter head is currently treated as a sum of independent low-frequency ( $v_{1i}$ ,  $v_{4i}$ ,  $v_{5i}$ ) and high-frequency ( $v_{2i}$ ,  $v_{3i}$ ) components governed by:

(a) The design features, namely, different number of picks engaged and, thus, different average cutting force on the cutter head. The design variation factor is determined from the expression:

$$v_{1i} = \frac{1}{F_{av}} \sqrt{\frac{1}{k} \sum_{j=1}^k (F_{avj} - F_{av})^2}, \quad (9)$$

where  $F_{av}$  is the average total cutting force on the cutter head;  $F_{avj}$  is the cutting force at a  $j$ -th position of the cutter head;  $k$  is the number of the positions (assumed as  $k \geq 32$ ).

(b) The brittle behavior of coal in fracture, described by the nonuniform (saw-like) form of the force diagram of a pick. In view of the actual arrangement of picks, the load variation factor of the cutter head is found from the formula:

$$v_{2i} = v_z \sqrt{\sum_{i=1}^{n_c} \left( \frac{Z_i}{F_{avmin}} \right)^2}, \quad (10)$$

where  $Z_i$  is the cutting force on each of  $i$  picks engaged in cutting, N;  $n_c$  is the number of picks on the cutter head;  $F_{avmin}$  is the minimum average value of the total cutting force on the cutter head.

The variation factor  $v_z$  of the cutting force on a pick is assumed as 0.5–1.2 depending on the coal seam structure.

(c) The varied cuttability  $A_{eqv}$  of coal in the section cut with the cutter head, which, together with the influence of the pick arrangement pattern on the cutter head, induces variation in load:

$$v_{3i} = v_{AC} \sqrt{\sum_{i=1}^{n_c} \left( \frac{Z_i}{F_{avmin}} \right)^2}. \quad (11)$$

The coefficient  $v_{AC}$  of cuttability variability in the transverse cross-section of coal face ranges from 0.47 at  $A_{eqv} < 120$  N/mm to 0.3 at  $A_{eqv} > 300$  N/mm.

(d) The varied cuttability  $A_{eqv}$  along the length of longwall, which causes low-frequency variation in load. The values of  $v_{4i}$  range between 0.24 at  $A_{eqv} < 120$  N/mm and 0.12 at  $A_{eqv} > 120$  N/mm.

(e) The varied loading due to nonuniform travel of shearer (plower). The variation factor  $v_{5i}$  (variation limit 0.35–0.05) grows with an increasing average load, lowers with an increasing rate of feed and linearly depends on the feed stiffness.

**Figure 2** shows the efficient operation zones for shearers with different capacity motors.

It is seen from the figure that efficient operation (maximal possible productivity) is governed by various constraints:

- maximal (stable) moment in the same structure seams and using motors with maximal test moments  $M_{\max.t} < 180$  dyne/m (Fig. 2a);
- durability of cutting drive motor in complex structure seams (groups 2 and 3) and using motors with  $M_{\max.t} < 180$  dyne/m (Fig. 2b);
- durability of the cutting drive transmission elements (at any coal breaking characteristics) and using motor with  $M_{\max.t} > 180$  dyne/m (Fig. 1c);
- allowable continuous (heat) power for shearers with motors of any capacity at duty cycle  $> 40\%$  (in high output longwalls) (Fig. 2d).

In coal seams with a dip more than  $20^\circ$ , it is possible that the line  $bc$  turns into a broken line  $eg$  (Fig. 2d) under the impact of the feed assembly pulling power constraint.

The operating modes inside or at the boundary of the field  $abcd$  ( $aegcd$ ) meet all constraints. The line  $bc$  in this case is the line of efficient operation at the highest productivity.

The governing factors of the efficient use of cutter heads are:

- the increase in face output through the increased productivity (feed rate) of mining machine owing to the reduced energy input of fracture at the full-scale utilization of the installed drive capacity;
- the increase in face output through the higher working capacity (cutting time ratio) as a result of shortened period of pick replacement and improved attachment of picks to cutter head;
- the adjustment of the electric energy cost with increasing (decreasing) energy consumed in mining;
- the adjustment of cutter head cost with increasing (decreasing) reliability and price;
- the decrease (worsening) of produced coal grade.

These factors should be considered as the direct sources of efficiency depending on the specificity of design and use of cutter heads.

The governing factors are interconnected. For the first turn, productivity depends on reliability of cutter heads. The actual practice shows that sometime intensification of cutting by means of mining machines with higher power-to-weight ratio produces no anticipated gain in productivity due to insufficient reliability of shearers and plowers and, first of all, their cutter heads. Design of mining machines lacks as a rule the highest efficiency in terms of coal grade and energy intake in favor of reliability of cutter heads and transmissions.

### Conclusion

The studies prove that efficiency evaluation of cutter heads is a complex multi-criterion problem requiring data on operating conditions, cutting mode, loads and reliability of mining machines and their components.

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